

## § 2.7 Linear Elasto Statics

$i, j, k, l$  are indicies associated with components. Repeated indicies sum to  $n$ sd

As with heat transfer we will make two passes through the weak form -  
 first: with indicial notation  
 second: with vectors and matrices

The transition between the two more complex because we

- will collapse 2<sup>nd</sup> order tensor's to vectors and a 4<sup>th</sup> order tensor to a matrix
- Change the definition of shear strain (just by a factor of 2)

Quantities of interest

$u_i$  - displacement components -  $u$  is now a vector  $\underline{u}$

$T_{ij}$  - Cauchy stress tensor components

$f_i$  - body force vector components

$\epsilon_{ij}$  - Green's strain tensor components

$$\epsilon_{ij} \equiv \underbrace{u_{(i,j)}}_{\text{symmetric derivative}} \equiv \frac{1}{2}(u_{i,j} + u_{j,i})$$

The stresses and strains are related through material parameters -  
Generalized Hooke's Law (Constitutive eq.)

$$\sigma_{ij} = C_{ijkl} \epsilon_{kl}$$

By definition  $\epsilon_{kl} = \epsilon_{lk}$  symmetric

By balance of angular momentum  
(equilibrium of stress block)

$$\sigma_{ij} = \sigma_{ji} \text{ symmetric}$$

This leads to symmetries in  $C_{ijkl}$

$$C_{ijkl} = C_{klij}$$

$$C_{ijkl} = C_{jike}$$

$$C_{ijke} = C_{ijlk}$$

$C_{ijkl}$  must also be pos. def. (at least for what we do)

$$C_{ijkl} \psi_{ij} \psi_{kl} \geq 0$$

$$C_{ijkl} \psi_{ij} \psi_{kl} = 0 \text{ iff } \psi_{ij} = \emptyset$$

↑  
all terms zero

## Strong form

Given  $f_i: \Omega \rightarrow \mathbb{R}$ ,  $g_i: \Gamma_{g_i} \rightarrow \mathbb{R}$  and  $h_i: \Gamma_{h_i} \rightarrow \mathbb{R}$   
 Find  $u_i: \Omega \rightarrow \mathbb{R}$  such that

$$\sigma_{ij,j} + f_i = 0 \quad \text{in } \Omega \quad (\text{equilibrium})$$

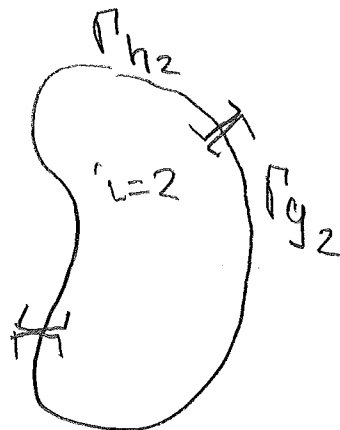
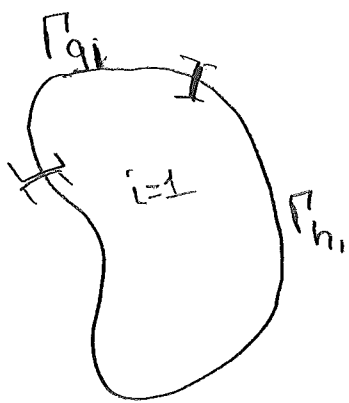
$$u_i = g_i \quad \text{on } \Gamma_{g_i} \quad \text{displacement B.C.}$$

$$\sigma_{ij} n_j = h_i \quad \text{traction B.C.}$$

where

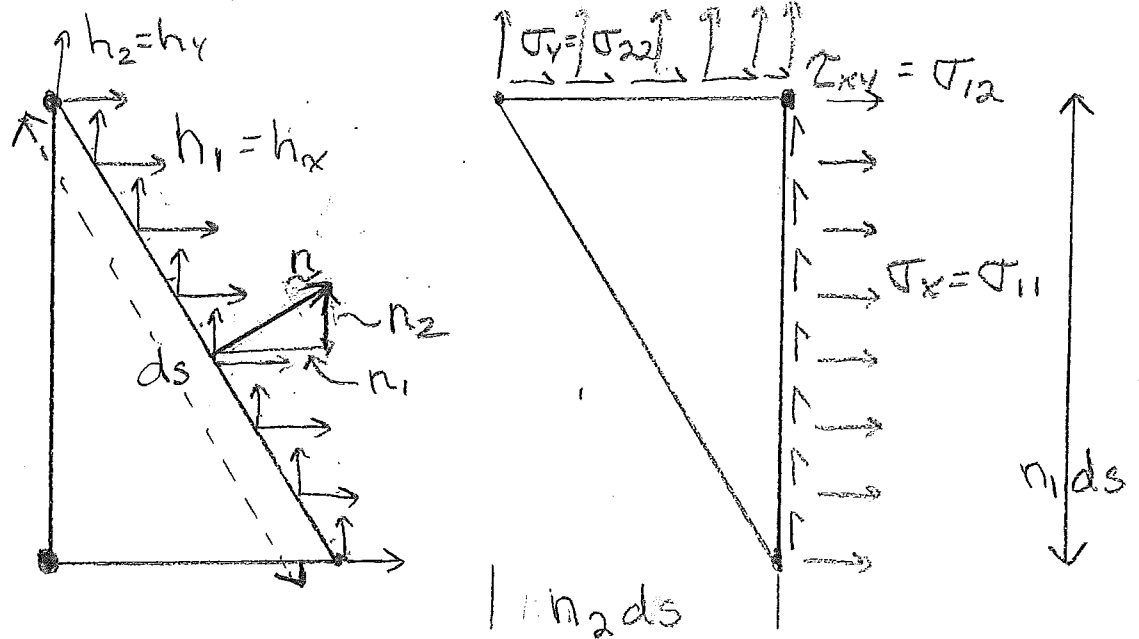
$$\epsilon_{ij} \equiv u_{(i,j)} \equiv \frac{u_{i,j} + u_{j,i}}{2}$$

$$\sigma_{ij} = C_{ijkl} \epsilon_{kl}$$



$$\Gamma = \bar{\Gamma}_{g_i} \cup \bar{\Gamma}_{h_i} \quad i=1(1) \text{ nsd}$$

$$O = \Gamma_{g_i} \cap \Gamma_{h_i} \quad i=1(1) \text{ nsd}$$



To see expressions for  $h_1$  and  $h_2$   
write statical equivalents

$$\sum F_x = 0$$

$$h_1 ds = \sigma_{11} n_1 ds + \sigma_{12} n_2 ds$$

$$h_1 = \sigma_{11} n_1 + \sigma_{12} n_2$$

$$\sum F_y = 0$$

$$h_2 ds = \sigma_{22} n_2 ds + \sigma_{12} n_1 ds$$

$$h_2 = \sigma_{22} n_2 + \sigma_{12} n_1$$

Applying MWR

$$\int_{\Omega} w_i (\sigma_{ij,j} - f_i) d\Omega = 0$$

see Lemma 2  
on pg 79 of  
text

integrate first term by parts

Because of symmetry of  $\sigma_{ij}$

$$-\int_{\Omega} w_{(i,j)} \sigma_{ij} d\Omega + \int_{\Gamma} w \sigma_{ij} n_j d\Gamma - \int_{\Omega} w_i f_i d\Omega = 0$$

$$\int_{\Gamma} w \sigma_{ij} n_j d\Gamma = \int_{\Gamma_{g_i}} w \sigma_{ij} n_j d\Gamma + \int_{\Gamma_{h_i}} w \sigma_{ij} n_j d\Gamma$$

$\rightarrow 0$  for  $w \in \mathcal{V}$

$$= \int_{\Gamma_{h_1}} w_1 \sigma_{2j} n_j d\Gamma + \int_{\Gamma_{h_2}} w_2 \sigma_{2j} n_j d\Gamma \quad (n_{sd}=2)$$

Since  $\Gamma_{h_1} \neq \Gamma_{h_2}$  in general - have two terms

Another way to write it is

$$\sum_{\alpha=1}^{n_{sd}} \left( \int_{\Gamma_{h_\alpha}} w_\alpha \sigma_{\alpha j} n_j d\Gamma \right) = \sum_{\alpha=1}^{n_{sd}} \left( \int_{\Gamma_{h_\alpha}} w_\alpha h_\alpha d\Gamma \right)$$

$\uparrow$   
no sum on  $\alpha$

Putting it together we can write the weak form -

Given  $f_i: \Omega \rightarrow \mathbb{R}$ ,  $g_i: \Gamma_{g_i} \rightarrow \mathbb{R}$  and  $h_i: \Gamma_{h_i} \rightarrow \mathbb{R}$ , find  $u_i \in \mathcal{S}_i$  such that

$$\int_{\Omega} w_{(i,j)} \nabla_{ij} d\Omega = \int_{\Omega} w_i f_i d\Omega + \sum_{\alpha=1}^{n_{sd}} \left( \int_{\Gamma_{h_i}} w_{\alpha} h_{\alpha} d\Gamma \right) \forall w_i \in \mathcal{V}_i$$

no sum in integrals

with  $\nabla_{ij} = C_{ijkl} \epsilon_{kl}$ ,  $\epsilon_{ij} = \epsilon_{(ij)} = \frac{u_{i,j} + u_{j,i}}{2}$

$$\mathcal{S}_i = \{u_i \mid u_i \in H^1, u_i = g_i \text{ on } \Gamma_{g_i}\} \quad \bar{i} = 1(1)n_{sd}$$

$$\mathcal{V}_i = \{w_i \mid w_i \in H^1, w_i = 0 \text{ on } \Gamma_{g_i}\} \quad \bar{i} = 1(1)n_{sd}$$

1. Review text discussion of  $(S) \rightarrow (W)$ ,  $(W) \rightarrow (S)$

note-  $\int_{\Omega} w_{(i,j)} \nabla_{ij} d\Omega = \int_{\Omega} w_{(i,j)} C_{ijkl} \epsilon_{kl} d\Omega$

Stated with abstract form -

Given  $f, g$  and  $h$  (defined over right stuff)  
 find  $\tilde{u} \in \tilde{\mathcal{S}}$  such that  
 $a(\tilde{w}, \tilde{u}) = (\tilde{w}, \tilde{f}) + (\tilde{w}, \tilde{h})_{\Gamma} \quad \forall \tilde{w} \in \tilde{\mathcal{V}}$

where

$$a(\tilde{w}, \tilde{u}) = \int_{\Omega} w_{(i,j)} C_{ijkl} \epsilon_{kl} d\Omega$$

$$(\tilde{w}, \tilde{f}) = \int_{\Omega} w_i f_i d\Omega$$

$$(\tilde{w}, \tilde{h})_{\Gamma} = \sum_{\alpha=1}^{n_{sd}} \int_{\Gamma_{h_{\alpha}}} w_{\alpha} h_{\alpha} d\Gamma$$

no sum.

We now work toward a "convenient" matrix description

we begin by representing the necessary terms from our second order tensors as terms listed in a vector -

(they still are second order tensors and transform coordinates like second order tensors - (not like vectors))

recalling that  $\sigma_{ij} = \sigma_{ji}$

for  $n_{sd} = 2$   $\sigma_{12} = \sigma_{21}$ , so only 3 independent terms  $\sigma_{11}, \sigma_{22}$  and  $\sigma_{12}$  (or  $\sigma_{21}$ )

Thus we will define

$$n_{sd}=2 \quad \underline{\sigma} = \begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{Bmatrix}$$

for  $n_{sd} = 3$  we have  $\sigma_{12} = \sigma_{21}, \sigma_{13} = \sigma_{31}, \sigma_{23} = \sigma_{32}$   
so  $\sigma_{11}, \sigma_{22}, \sigma_{33}, \sigma_{12}, \sigma_{13}$  &  $\sigma_{23}$  can be the 6 independent terms

The text book ordering of the 6 terms

$$n_{sd}=3 \quad \underline{\sigma} = \begin{Bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{Bmatrix}$$

Like wise we define a  $3 \times 1$  strain vector for  $n_{sd}=2$  and  $6 \times 1$  strain vector for 3D  
 How we convert from Green's strains to "Engineering strains" (double "shear" terms).

$$n_{sd}=2 \quad \underline{\underline{E}}(u) = \{ \underline{\underline{E}}_I(u) \} = \left\{ \begin{array}{l} u_{1,1} \\ u_{2,2} \\ u_{1,2} + u_{2,1} \end{array} \right\}$$

$$n_{sd}=3 \quad \underline{\underline{E}}(u) = \{ \underline{\underline{E}}_I(u) \} = \left\{ \begin{array}{l} u_{1,1} \\ u_{2,2} \\ u_{3,3} \\ u_{2,3} + u_{3,2} \\ u_{1,3} + u_{3,1} \\ u_{1,2} + u_{2,1} \end{array} \right\}$$

We need our constitutive eq. in terms of the new stress and strain vectors

$$\underline{\underline{\sigma}} = \underline{\underline{D}} \underline{\underline{E}}(u) \quad , \quad \{ \underline{\underline{\sigma}} \}_{ex1} = [D] \{ \underline{\underline{E}}(u) \}_{ex1}$$

$$[D] = [D]^T \text{ symmetric}$$

$$e = n_{sd}(n_{sd}+1)/2 \quad , \quad e = 3 \text{ for } n_{sd}=2 \quad , \quad e = 6 \text{ for } n_{sd}=3$$

If we are given the terms  $C_{ijkl}$  using the known symmetries of  $C_{ijkl}$  and  $\underline{\underline{D}}$  as well as the relations from  $\sigma_{ij}$  to  $\underline{\underline{\sigma}}$  and  $\underline{\underline{E}}_{ij}$  to  $\underline{\underline{E}}(u)$  we can construct terms in  $\underline{\underline{D}}$ .  
 Text book shows how to do this  
 Since no one does this - I will not cover it or ask you to do it on homework or test.



with all this done we have

$$w_{(i,j)} C_{ijkl} u_{(k,l)} = \int_{\Omega} \underline{\underline{w}}(\underline{\underline{w}})^T \underline{\underline{D}} \underline{\underline{e}}(\underline{\underline{u}}) d\Omega$$

$$a(\underline{\underline{w}}, \underline{\underline{u}}) = \int_{\Omega} \underline{\underline{e}}(\underline{\underline{w}})^T \underline{\underline{D}} \underline{\underline{e}}(\underline{\underline{u}}) d\Omega$$

Doing our normal decomposition of  $\underline{\underline{u}}^h = \underline{\underline{v}}^h + \underline{\underline{q}}^h$ ,  $\underline{\underline{u}}^h \in \underline{\underline{\delta}}^h$ ,  $\underline{\underline{v}}^h \in \underline{\underline{V}}^h$ ,  $\underline{\underline{q}}^h \in \underline{\underline{\delta}}^h$

where:  
 $\underline{\underline{V}}^h = \{ \underline{\underline{w}}^h \mid w_i^h \in \underline{\underline{V}}_i^h \}$ ,  $\underline{\underline{V}}_i^h = \{ w_i^h \mid w_i^h \in H^1, w_i^h = 0 \text{ on } \Gamma_{g_i} \}$   
 $\underline{\underline{\delta}}^h = \{ \underline{\underline{u}}^h \mid u_i^h \in \underline{\underline{\delta}}_i^h \}$ ,  $\underline{\underline{\delta}}_i^h = \{ u_i^h \mid u_i^h \in H^1, u_i^h = g_i \text{ on } \Gamma_{g_i} \}$

we have our Galerkin form

Given  $\underline{\underline{f}}, \underline{\underline{g}}$  and  $h$  as before find  $\underline{\underline{u}}^h = \underline{\underline{v}}^h + \underline{\underline{q}}^h \in \underline{\underline{\delta}}^h$  such that

$$a(\underline{\underline{w}}^h, \underline{\underline{v}}^h) = (\underline{\underline{w}}^h, \underline{\underline{f}}) + (\underline{\underline{w}}^h, \underline{\underline{h}})_{\Gamma} - a(\underline{\underline{w}}^h, \underline{\underline{g}}^h)$$

To get to the matrix form we have to deal with the fact that essential BC can be associated with any nodes (in general dof holders)

(n dof  $\geq$  Nsd for vector systems)

Assuming nodal dof only where each node can have  $n_{dof}$ , lets use  $n_{dof} = N_{sd}$  for the basic vector system

in this case the total possible number of dof is

$$n_{\text{DOF}} = n_{\text{dof}} n_{\text{np}} = n_{\text{sd}} n_{\text{np}} \quad \text{for our current case}$$

⏟ # of nodes

$n_{\text{eq}}$  = actual # dof in global system

$n_{\text{eq}} < n_{\text{DOF}} \Leftarrow$  must have sufficient essential B.C.

Consider again the set  $\eta = \{1, 2, 3, \dots, n_{\text{np}}\}$   
 and now define set

$$n_{q_i} = \{\text{nodes with a } q_i\}$$

with this we can write

$$v_i^h = \sum_{A \in n_{q_i}} N_A d_{iA} \quad i = 1(1)n_{\text{sd}}$$

$$q_i = \sum_{A \in n_{q_i}} N_A q_{iA} \quad i = 1(1)n_{\text{sd}}$$

Introducing yet another construct:

Euclidean basis vector  $e_i$

$$n_{\text{sd}} = 2 \quad \underset{\sim}{e}_1 = \begin{Bmatrix} 1 \\ 0 \end{Bmatrix}, \quad \underset{\sim}{e}_2 = \begin{Bmatrix} 0 \\ 1 \end{Bmatrix}$$

$$n_{\text{sd}} = 3 \quad \underset{\sim}{e}_1 = \begin{Bmatrix} 1 \\ 0 \\ 0 \end{Bmatrix}, \quad \underset{\sim}{e}_2 = \begin{Bmatrix} 0 \\ 1 \\ 0 \end{Bmatrix}, \quad \underset{\sim}{e}_3 = \begin{Bmatrix} 0 \\ 0 \\ 1 \end{Bmatrix}$$

with this we can write

$$\underline{v}^h = v_i^h \underline{e}_i, \quad \underline{g}^h = g_i^h \underline{e}_i \quad \left( \begin{array}{l} \text{remember to} \\ \text{sum} \end{array} \right)$$

$$\underline{w}^h = w_i^h \underline{e}_i, \quad w_i^h = \sum_{A \in \mathcal{N} - \mathcal{N}_g} N_A C_{iA}$$

Note - notation still a bit messy but idea is to include the correct terms in each summation as well as accounting for components -

Substituting all this in and recalling the  $C_{iA}$ 's are arbitrary we have

$$\sum_{j=1}^{n_{sd}} \left( \sum_{B \in \mathcal{N} - \mathcal{N}_g} a(N_A \underline{e}_i, N_B \underline{e}_j) \right) d_{jB} =$$

(Using the dof =  $n_{sd}$  case)

$$= \sum_{j=1}^{n_{sd}} \left( \sum_{B \in \mathcal{N}_g} a(N_A \underline{e}_i, N_B \underline{e}_j) \right) \underline{g}_{jB} + (N_A \underline{e}_i, \underline{f}) + (N_A \underline{e}_i, \underline{h}) \Gamma$$

$A \in \mathcal{N} - \mathcal{N}_g$   
 $1 \leq i \leq n_{sd}$

For writing the final matrix form

we can use the ID device - now as a matrix

$$ID(i, A)$$

$$i = 1 \dots n_{sd}$$

$$A = 1 \dots n_{np}$$

(vector system nodal dof only)

P=0

Do A=1 to N<sub>np</sub>

Do i=1 to N<sub>sd</sub>

If A ∈ N - N<sub>gi</sub>

P=P+1

ID(i, A) = P ← equation # in K

else

ID(i, A) = 0

end if

end Do

end Do

N<sub>eq</sub> = P ← total number of actual equations

Using this we have

$$[K] \{d\} = \{F\} \quad \left( \underset{\sim}{K} \underset{\sim}{d} = \underset{\sim}{F} \right)$$

$$P = ID(i, A), \quad Q = ID(j, B)$$

$$K_{PQ} = a(N_A \underset{\sim}{e}_i, N_B \underset{\sim}{e}_j)$$

$$F_P = (N_A \underset{\sim}{e}_i, \underset{\sim}{f}) + (N_A \underset{\sim}{e}_i, \underset{\sim}{h}) \Gamma$$

$$- \sum_{j=1}^{N_{sd}} \left( \sum_{B \in N_{g_j}} a(N_A \underset{\sim}{e}_i, N_B \underset{\sim}{e}_j) g_{jB} \right)$$

⇒ We will see exactly how terms go into place when we locate our pseudo code later

To complete the ability to use vector matrix form we want to introduce the "B" matrix again

$$\underline{\underline{E}}(\underline{N}_A \underline{e}_i) = \underline{B}_A \underline{e}_i \Leftarrow \text{Strain-displacement operator}$$

for  $n_{sd}=2$

$$\underline{B}_A = \begin{bmatrix} N_{A,1} & 0 \\ 0 & N_{A,2} \\ N_{A,2} & N_{A,1} \end{bmatrix}$$

$3 \times 2$   
 $\uparrow \quad \nwarrow$   
 $\# \text{ strain components} \quad n_{sd}$

for  $n_{sd}=3$

$$\begin{bmatrix} N_{A,1} & 0 & 0 \\ 0 & N_{A,2} & 0 \\ 0 & 0 & N_{A,3} \\ 0 & N_{A,3} & N_{A,2} \\ N_{A,3} & 0 & N_{A,1} \\ N_{A,2} & N_{A,1} & 0 \end{bmatrix}$$

$6 \times 3$

with this we can write (not very useful!)

$$K_{pq} = \underline{e}_i^T \int \underline{B}_A^T \underline{D} \underline{B}_B d\Omega \underline{e}_j$$

and noting that  $\int_{\Omega} N_A \underline{e}_i \underline{e}_j d\Omega = \int_{\Omega} N_A f_{ij} d\Omega$

$$\int_{\Gamma} N_A \underline{e}_i h d\Gamma = \int_{\Gamma_{hi}} N_A h_i d\Gamma$$

$$F_p = \int_{\Omega} N_A f_i d\Omega + \int_{\Gamma_{hi}} N_A h_i d\Gamma - \underbrace{\sum_{j=1}^{N_{sd}} \left( \sum_{B \in \mathcal{N}_{g_j}} a(N_A e_i, N_B e_j) \right) g_{jB}}_{\text{still messy -}}$$

Is  $K_{PA} = K_{QP}$  (symmetric)

$$K_{PA} = \left( e_i^T \int_{\Omega} \tilde{B}_A^T D \tilde{B}_B d\Omega e_j \right)^T \leftarrow \begin{matrix} \text{since its} \\ \text{just a} \\ \text{"number"} \end{matrix}$$

$$= e_j^T \int_{\Omega} \tilde{B}_B^T D^T \tilde{B}_A e_i \quad \text{with } D = D^T$$

$$= e_j^T \int_{\Omega} \tilde{B}_B^T D \tilde{B}_A e_i \equiv K_{QP}$$

Positive Definiteness of  $K$

$$\tilde{c}^T K \tilde{c} \geq 0, \quad \tilde{c}^T K \tilde{c} = 0 \text{ iff } \tilde{c} = \emptyset$$

$$\tilde{c}^T K \tilde{c} = \sum_{P, Q=1}^{n_{eg}} C_P K_{PQ} C_Q$$

$$= \sum_{i, j=1}^{n_{dof}} \left( \sum_{A \in \mathcal{N}_{-n_{g_i}}} C_{iA} a(N_A e_i, N_B e_j) C_{jB} \right)$$

Use bilinearity

$$= a \left( \sum_{i=1}^{n_{dof}} \left( \sum_{A \in \mathcal{N}_{-n_{g_i}}} C_{iA} N_A e_i \right), \sum_{j=1}^{n_{dof}} \left( \sum_{B \in \mathcal{N}_{-n_{g_j}}} C_{jB} N_B e_j \right) \right)$$

$$= a(w^h, w^h) = \int_{\Omega} \underbrace{w_{(i,j)}^h C_{ij}^h w_{(k,e)}^h}_{\geq 0 \text{ for pos def } C_{ij}^h} d\Omega$$

we have positive ness

assume  $\underbrace{C^T}_{\sim} K \underbrace{C}_{\sim} = 0$

That requires  $\omega_{(i,j)}^h C_{ijke} \omega_{(ke)}^h = 0$

This requires  $\omega_{(i,j)}^h = 0$  for pos. def.  $C_{ijke}$

There are a limited set of  $\omega_{\sim}^h \neq 0$  for which  $\omega_{(i,j)}^h = \frac{\omega_{ij}^h + \omega_{ji}^h}{2} = 0$

The correspond

for  $n_{sd}=2$   $\omega_{\sim}^h(x) = C_{\sim} + C_3 (x_1 e_2 - x_2 e_1)$   
 rigid body translation      rigid body rotation in plane

for  $n_{sd}=3$   $\omega_{\sim}^h(x) = C_1 + C_2 \times \times \leftarrow$  Rigid body rotations  
 rigid body translation      cross product

$\underbrace{K}_{\sim}$  is pos.-def. so long as rigid body motions are constrained by the prescribed  $\omega_{\sim}^h$

Finally - We do not want to do those crazy integrals to directly get  $k_{pq}$  and  $F_p$  terms

We want to do elements - the whole element - at once and have the assembly operator take care of putting things in the right place the right way

$$\tilde{K} = \sum_{e=1}^{nel} A \tilde{k}^e, \quad \tilde{F} = \sum_{e=1}^{nel} A (\tilde{k}^e, \tilde{f}^e)$$

$N_{en} = \# \text{ nodes / element}$

$N_{ed} = \# \text{ dof / node} = N_{sd}$  for simple vector systems

$N_{ee} = N_{en} N_{ed} = \# \text{ of dof of the element}$   
 $\rightarrow$  will be the size of element  $\tilde{k}^e, \tilde{f}^e$

$$\tilde{k}^e = [k_{pq}^e], \quad \tilde{f}^e = \{f^{ej}\} \quad 1 \leq p, q \leq N_{ee}$$

$$k_{pq} = \int_{\Omega_e} e_i^T B_a^T D B_b dr e_j \quad \begin{matrix} p = N_{ed}(a-1) + i \\ q = N_{ed}(b-1) + j \end{matrix}$$

for  $N_{sd} = 2$

$$\tilde{B}_a = \begin{bmatrix} N_{a,1} & 0 \\ 0 & N_{a,2} \\ N_{a,2} & N_{a,1} \end{bmatrix}$$

for

$N_{sd} = 3$

$$\tilde{B}_a =$$

$$\begin{bmatrix} N_{a,1} & 0 & 0 \\ 0 & N_{a,2} & 0 \\ 0 & 0 & N_{a,3} \\ 0 & N_{a,3} & N_{a,2} \\ N_{a,3} & 0 & N_{a,1} \\ N_{a,2} & N_{a,1} & 0 \end{bmatrix}$$



what I called  $\tilde{f}_p^e$

$$f_p^e = \int_{\Omega^e} N_a f_i d\Omega + \int_{\Gamma_{hi}^e} N_a h_i d\Gamma$$

$$- \sum_{q=1}^{n_{ee}} k_{pq} g_q^e$$

$\Gamma_{hi}^e = \Gamma_{hi} \cap \Omega^e$   
 ↑ portion of elements

$g_q = g_j(x_j^e)$  if node is on  $\Gamma_j$   
 is zero otherwise

In actual programs we avoid the  $e$ 's and just do the whole thing at once for example -

Define  $\tilde{B} = [\tilde{B}_1, \tilde{B}_2, \dots, \tilde{B}_{n_{en}}]$

then  $\tilde{k}^e = \int_{\Omega^e} \tilde{B}^T \tilde{D} \tilde{B} d\Omega$

Stress evaluation  $\tilde{\epsilon}(u^e) = \sum_{a=1}^{n_{en}} \tilde{B}_a d_a$   $\leftarrow$  dof at a node

$$\tilde{\sigma}(x) = \tilde{D}(x) \tilde{\epsilon}(u^e(x)) = \tilde{D}(x) \sum_{a=1}^{n_{en}} \tilde{B}_a d_a$$

$$= \tilde{D}(x) \tilde{B} d^e$$

where

$$\tilde{B} = [\tilde{B}_1, \tilde{B}_2, \tilde{B}_3, \dots, \tilde{B}_{n_{en}}], \quad d = \begin{Bmatrix} d_1^e \\ d_2^e \\ \vdots \\ d_{n_{en}}^e \end{Bmatrix}$$

Before seeing how this all fits together in the form of pseudo code to a very basic F.E. code, we need one more tool - Numerical Integration (will cover Chapt. 3)

$$\underline{k}^e = \int_{\Omega} \underline{B}^T \underline{D} \underline{B} d\Omega \quad \text{is often messy because of mappings-}$$

Thus we will apply numerical integration

$$\underline{k}^e = \sum_{l=1}^{n_{int}} w_l \underline{B}^T(\underline{x}_e) \underline{D}(\underline{x}_e) \underline{B}(\underline{x}_e) + R^{int}$$

Since we do not know  $R^{int}$  past being able to say it small enough when we use a sufficient number of properly selected integration points, we will drop it and use

$$\underline{k}^e \approx \sum_{l=1}^{n_{int}} w_l \underline{B}^T(\underline{x}_e) \underline{D}(\underline{x}_e) \underline{B}(\underline{x}_e)$$